

## Specification

## HEAT EXCHANGER AND AIR CONDITIONER INDOOR UNIT

## (TECHNICAL FIELD)

The present invention relates to a heat exchanger, and more particularly to  
5 a heat exchanger that is disposed in an indoor unit of an air conditioner, and an  
indoor unit of an air conditioner.

## (BACKGROUND ART)

Heat exchangers that are formed by combining a plurality of heat  
exchange units at an angle are conventionally used. This type of heat  
10 exchanger can be formed into a variety of shapes that meet design needs and  
the like by combining heat exchange units. Many of the plurality of heat  
exchange units that form the heat exchanger have various lengths (see  
Japanese Unexamined Patent Publication No. 2001-4162). For example, there  
is a heat exchanger that makes up a portion of an indoor unit of an air  
15 conditioner that is formed into an inverted V shape so as to surround a ventilation  
fan. This heat exchanger is a combination of heat exchange units of various  
lengths which are formed into an inverted V shape, and the heat exchange units  
that cover the front of the ventilation fan and heat exchange units that cover the  
rear of the ventilation fan have different lengths.

20 When a heat exchanger formed by means of a plurality of heat exchange  
units that are angled and combined together is assembled, errors in the  
attachment angle of the heat exchange units will create problems. In other  
words, when there are errors in the attachment angle of the heat exchange units,  
errors will be produced in the positions of the end portions of the heat exchanger,  
25 and thus problems may be produced in the positioning and the like of the heat  
exchanger. Thus, errors in the attachment angle of the heat exchange units are  
preferably as small as possible, but if a very severe error tolerance is required,  
the ability to assemble the heat exchanger will be reduced.

On the other hand, errors in the position of the end portions of the heat

exchanger will become larger as the length of the heat exchange units become longer, even if the errors in the attachment angle of the heat exchange units are the same. Because of this, it is thought that the error tolerance of the attachment angle can be relaxed by shortening the lengths of the heat exchange units.

However, if the lengths of the heat exchange units are shortened, there will be an impact on the heat exchangeability of the heat exchanger. In other words, if the lengths of the heat exchange units are shortened, the overall surface area of the heat exchanger (the sum of the surface area of each heat exchange unit) will decrease, and the heat exchangeability of the heat exchanger will be reduced. Thus, simply shortening the lengths of the heat exchange units in order to relax the error tolerance of the attachment angle of the heat exchange units will give rise to a reduction in heat exchangeability.

#### (DISCLOSURE OF THE INVENTION)

An object of the present invention is to provide a heat exchanger and an indoor unit of an air conditioner that can both suppress a reduction in heat exchangeability and relax the error tolerance of the attachment angle of the heat exchange units.

The heat exchanger disclosed in claim 1 is a heat exchanger that is formed by connecting a plurality of heat exchange units, is disposed in an indoor unit of an air conditioner, and includes a first heat exchange unit, a second heat exchange unit, and a third heat exchange unit. The second heat exchange unit is connected at an angle with one end of the first heat exchange unit. The third heat exchange unit is connected at an angle with the other end of the first heat exchange unit. Then, the second heat exchange unit and the third heat exchange unit will have approximately the same length.

With this heat exchanger, the second heat exchange unit and the third heat exchange unit have approximately the same length. When the total length of heat exchangers are the same, the heat exchanger in which the lengths of the

second heat exchange unit and the third heat exchange unit are the same will have a smaller maximum amount of error in the positions of the end portions of the heat exchanger due to error in the attachment angle than the heat exchanger in which the lengths of the second heat exchange unit and the third heat exchange unit are different. In other words, when the lengths of the second heat exchange unit and the third heat exchange unit are different, one of the heat exchange units will be longer and the other will be shorter. In this situation, position error in the end portions of the heat exchanger due to the long heat exchange unit will increase. On the other hand, if the total length of the heat exchanger is the same, the lengths of the heat exchange units when the lengths of the second heat exchange unit and the third heat exchange unit are the same will be shorter than the length of the long heat exchange unit when the lengths are different. Thus, with this heat exchanger, a reduction in heat exchangeability can be suppressed and the error tolerance of the attachment angle of the heat exchange units can be relaxed.

The heat exchanger disclosed in claim 2 is the heat exchanger disclosed in claim 1, in which the first heat exchange unit has an approximate inverted V shape in cross-section. Then, the second heat exchange unit and the third heat exchange unit respectively extend downward from the front and rear lower ends of the first heat exchange unit.

When the lower ends of the second heat exchange unit and the third heat exchange unit are the respective lower ends of the heat exchanger, errors in the attachment angle of the second heat exchange unit and the third heat exchange unit will have an impact on the position error of the lower ends of the heat exchanger.

However, with this heat exchanger, the lengths of the second heat exchange unit and the third heat exchange unit are approximately the same, and thus the maximum amount of error in the positions of the lower ends of the heat exchanger due to error in the attachment angle will decrease. Thus, with this

heat exchanger, the error tolerance of the attachment angle of the second heat exchange unit and the third heat exchange unit can be relaxed.

The heat exchanger disclosed in claim 3 is the heat exchanger disclosed in claim 1 or claim 2, in which the heat exchanger is symmetrical from front to rear, and the second heat exchange unit and the third heat exchange unit are symmetrical from front to rear.

With this heat exchanger, the lengths of the second heat exchange unit and the third heat exchange unit are approximately the same because the second heat exchange unit and the third heat exchange unit are symmetrical from front to rear. Thus, the error tolerance of the attachment angle of the second heat exchange unit and the third heat exchange unit can be relaxed.

An indoor unit of an air conditioner disclosed in claim 4 includes the heat exchanger disclosed in any of claims 1 to 3, and a ventilation fan that is disposed so as to be covered by the heat exchanger.

With this indoor unit of an air conditioner, the ventilation fan is disposed so as to be covered by the heat exchanger. Thus, in order to maintain the distance between the heat exchanger and the ventilation fan at a predetermined value, the precision of the distance between the heat exchanger and the ventilation fan is important. Because of this, the shape of the heat exchanger preferably has a high degree of accuracy. Thus, the present invention, which can relax the error tolerance of the attachment angle between the second heat exchange unit and the third heat exchanger unit, is very useful.

The indoor unit of an air conditioner disclosed in claim 5 includes a ventilation fan, a heat exchanger, a first drain pan, a second drain pan, and a drain path. The heat exchanger is the heat exchanger disclosed in claim 1, covers the front, upper and rear portions of the ventilation fan, and is disposed so that the lower front end and the lower rear end of the heat exchanger are at the height of the apex of the ventilation fan or lower. The first drain pan is disposed below the lower front end of the heat exchanger. The second drain

pan is disposed below the lower rear end of the heat exchanger. Drain water that is discharged from the first drain pan and the second drain pan passes through the drain path. Then, the first drain pan and the second drain pan are disposed at approximately the same height.

5           Generally, in an indoor unit of an air conditioner that includes a heat exchanger, the heat exchanger will function as a vaporizer during cooling and the like, and thus moisture in the air will condense on the surface of the heat exchanger and generate drain water. Because of this, the indoor unit of an air conditioner will normally include a drain pan that receives drain water. This  
10 drain pan will normally be disposed below the heat exchanger in order to receive the drain water that drips down from the heat exchanger. Thus, when the heat exchanger is disposed so as to cover the front and rear portions of the ventilation fan, drain pans will be respectively disposed below the lower front end and the lower rear end of the heat exchanger. In this configuration, the height at which  
15 the front drain pan and the rear drain pan are disposed is often different (see Japanese Unexamined Patent Publication No. 2000-74409). For example, the front drain pan is disposed lower and the rear drain pan is disposed higher, or the front drain pan is disposed higher and the rear drain pan is disposed lower.

          The drain water that drips down into the drain pans is discharged to the  
20 exterior of the indoor unit from an outlet of the drain pans via a drain path. In this configuration, the bigger the difference in height between the position of the drain pans and the position of the drain path, the more efficient the discharge of the drain water will be.

          On the other hand, from the point of view of reducing the height of the  
25 indoor unit, the heat exchanger is often brought close to the ventilation fan, and disposed so that the lower ends of the heat exchanger are lower than the apex of the ventilation fan. Thus, if the positions of the lower ends of the heat exchanger are lowered, the positions of the drain pans will also be lowered. Because of this, the difference in height of the drain pans and the drain path will

be smaller, and it will be difficult to discharge the drain water efficiently.

In addition, the upward movement of the drain pans is limited because the drain pans are disposed below the heat exchanger. Thus, if there is a difference in height between the first drain pan and the second drain pan, the position of one of the drain pans will be lower. Because of this, the difference in height of the positions of the drain path and the drain pans will be smaller.

However, with this indoor unit of an air conditioner, the first drain pan and the second drain pan that are disposed below the lower ends of the heat exchanger are disposed at approximately the same height. Thus, the lowering of one of the drain pans can be suppressed. Thus, with this indoor unit of an air conditioner, the difference in height of the drain path that discharges drain water and the drain pans can be largely ensured.

The indoor unit of an air conditioner disclosed in claim 6 is the indoor unit of an air conditioner disclosed in claim 5, in which the heat exchanger has an approximate inverted V shape in cross-section.

With this indoor unit of an air conditioner, the heat exchanger has an approximate inverted V shape in cross-section. Because of this, by disposing the ventilation fan in a space that is surrounded by the inverted V shaped heat exchanger, the front, upper and rear portions of the ventilation fan will be covered, and the lower ends of the heat exchanger can be easily disposed so as to be lower than the apex of the ventilation fan. Thus, the indoor unit of an air conditioner can be reduced in size in the height direction.

Note that the heat exchanger not only has a cross-sectional shape consisting only of the approximate inverted V shape portion, but may also have a cross-sectional shape that is formed by the approximate inverted V shaped portion and portions that extend downward from both lower ends thereof.

The indoor unit for an air conditioner disclosed in claim 7 is the indoor unit of an air conditioner disclosed in claim 5 or claim 6, in which the lower front end and the lower rear end of the heat exchanger are positioned at approximately the

same height.

With this indoor unit of an air conditioner, the lower front end and the lower rear end of the heat exchanger are positioned at approximately the same height. Then, the first drain pan and the second drain pan are respectively disposed  
5 below the lower front end and the lower rear end of the heat exchanger. Thus, with this indoor unit of an air conditioner, even if the first drain pan and the second drain pan are disposed in positions close to the lower ends of the heat exchanger, the first drain pan and the second drain pan can be disposed at approximately the same height.

10 The indoor unit of an air conditioner disclosed in claim 8 is the indoor unit of an air conditioner disclosed in any of claims 5 to 7, in which the heat exchanger has a shape that is symmetrical from front to rear.

With this indoor unit of an air conditioner, the heat exchanger has a shape that is symmetrical from front to rear. Thus, the heat exchanger is shaped so  
15 that the lower front end and the lower rear end thereof will be at the same height. Because of this, even if the first drain pan and the second drain pan are disposed in positions close to the heat exchanger, the first drain pan and the second drain pan can be disposed at approximately the same height.

#### BRIEF DESCRIPTION OF THE DRAWINGS

20 Fig. 1 is an exterior view of an air conditioner.

Fig. 2 is schematic diagram of a refrigerant circuit.

Fig. 3(a) is a front view of an indoor unit.

Fig. 3(b) is a right side view of the indoor unit.

Fig. 4 is a right side view of the indoor unit with an upper casing removed.

25 Fig. 5 is a right side cross-sectional view of the indoor unit.

Fig. 6 is a plan view of a right side portion of the indoor unit with the upper casing removed therefrom.

Fig. 7 is a right side view of a lower unit.

Fig. 8 is a plan view of a right side portion of the lower unit.

Fig. 9 is a right side cross-sectional view of the lower unit.

Fig. 10(a) is a lateral cross-sectional view of an indoor heat exchanger.

Fig. 10(b) is a lateral cross-sectional view of an assumed indoor heat exchanger.

5 Fig. 11(a) is an enlarged schematic view of a front end of the indoor heat exchanger.

Fig. 11(b) is an enlarged schematic view of a front end of the assumed indoor heat exchanger.

10 Fig. 12 is a lateral cross-sectional view of an indoor heat exchanger according to another embodiment.

(BEST MODE FOR CARRYING OUT THE INVENTION)

[Overall structure of the air conditioner]

The exterior of an air conditioner in which an embodiment of the present invention has been adapted is shown in Fig 1.

15 The air conditioner 1 includes an indoor unit 2 that is installed on an indoor wall surface or the like, and an outdoor unit 3 that is disposed outdoors.

An indoor heat exchanger 50 is accommodated in the indoor unit 2, an outdoor heat exchanger 30 is accommodated in the outdoor unit 3, and a refrigerant circuit is formed by connecting each heat exchanger 30, 50 by means  
20 of a refrigerant line 4.

[Schematic configuration of the air conditioner refrigerant circuit]

The configuration of a refrigerant circuit of the air conditioner 1 is shown in Fig. 2. The refrigerant circuit is formed primarily of the indoor heat exchanger 50, an accumulator 31, a compressor 32, a four-way switching valve 33, an  
25 outdoor heat exchanger 30, and an electric expansion valve 34.

The indoor heat exchanger 50 that is provided in the indoor unit 2 exchanges heat with the air that it comes into contact with. In addition, a cross flow fan 71 that serves to discharge air indoors after indoor air is taken in, passed through the indoor heat exchanger 50, and heat exchange has taken



place, is provided in the indoor unit 2. This cross flow fan 71 is formed into a long, narrow cylindrical shape, and is disposed so that the central axis thereof is parallel in the horizontal direction. The cross flow fan 71 is rotatably driven by means of an indoor fan motor 72 that is provided inside the indoor unit 2. The detailed structure of the indoor unit 2 will be described below.

A compressor 32, a four-way switching valve 33 that is connected to the discharge side of the compressor 32, an accumulator 31 that is connected to the intake side of the compressor 32, an outdoor heat exchanger 30 that is connected to the four-way switching valve 33, and an electric expansion valve 34 that is connected to the outdoor heat exchanger 30, are provided in the outdoor air conditioning unit 3. The electric expansion valve 34 is connected to a line 41 via a filter 35 and a liquid shut off valve 36, and is connected to one end of the indoor heat exchanger 50 via the line 41. In addition, the four way directional control valve 33 is connected to a line 42 via a gas shut off valve 37, and is connected to the other end of the indoor heat exchanger 50 via this line 42. These lines 41, 42 correspond to the refrigerant line 4 shown in Fig. 1. In addition, a propeller fan 38 that serves to discharge air outside after heat exchange with the outdoor heat exchanger 30 is provided in the outdoor unit 3. The propeller fan 38 is rotatably driven by an outdoor fan motor 39.

[Structure of the indoor unit]

A front view of the indoor unit is shown in Fig. 3(a), and a side view of the indoor unit 2 is shown in Fig. 3(b). The indoor unit 2 is rectangular in shape in the horizontal direction when viewed from the front, and has a vertical two tone color scheme when viewed from the front and from the sides.

The indoor unit 2 is formed primarily by an upper casing 6, a lower unit 7, and an indoor heat exchanger unit 5 that is accommodated in the interior of the indoor unit 2. The upper casing 6 covers the upper portion of the indoor unit 2. The lower unit 7 forms the lower portion of the indoor unit 2. The upper casing 6 and the lower unit 7 are formed into separate members, and the border

between the upper casing 6 and a portion of the lower unit 7 appears as a horizontal line on the exterior of the indoor unit 2. In addition, the upper casing 6 and a portion of the lower unit 7 are different colors, and the horizontal line that is the border between the upper casing 6 and the lower unit 7 forms the vertical two color scheme.

Each structure of the indoor unit 2 will be described below.

<Indoor heat exchange unit>

As shown in Fig. 4, the indoor heat exchange unit 5 is formed by the indoor heat exchanger 50, auxiliary lines 51, an auxiliary support member 52, and the like. Note that Fig. 4 is a right side view of the indoor unit 2 with the upper casing 6 removed.

Fig. 5 shows a cross-sectional side view of the indoor unit 2.

The indoor heat exchanger 50 is installed so as to surround the front, upper, and rear portions of the cross flow fan 71, and air drawn in from intake ports 601, 611 by rotating the cross flow fan 71 passes through the indoor heat exchanger 50 to the cross flow fan 71, and heat exchange is performed with the refrigerant that passes through the interior of a heat transfer line. The indoor heat exchanger 50 is divided into four components: a first indoor heat exchanger 50a, a second indoor heat exchanger 50b, a third indoor heat exchanger 50c, and a fourth indoor heat exchanger 50d. By connecting each of the indoor heat exchangers 50a, 50b, 50c, and 50d, the indoor heat exchanger 50 is formed so as to have an approximate inverted V-shape in cross-section whose both ends are bent downward when viewed from the sides.

Each of the indoor heat exchangers 50a, 50b, 50c, and 50d have a long plate shape in the horizontal direction. Each of the indoor heat exchangers 50a, 50b, 50c, and 50d are formed from a heat transfer line that is folded back a plurality of times at both ends thereof, and a plurality of fins having short strip shapes in which the heat transfer line is inserted. The heat transfer line is folded back by means of U-shaped heat transfer lines at both side ends of each

of the indoor heat exchangers 50a, 50b, 50c, and 50d.

The upper end of the first indoor heat exchanger 50a is inclined toward the front of the indoor unit 2, and the first indoor heat exchanger 50a is disposed so as to cover the cross flow fan 71 from the upper central portion thereof to the  
5 upper rear side thereof.

The upper end of the second indoor heat exchanger 50b is inclined toward the rear of the indoor unit 2, and the second indoor heat exchanger 50b is disposed forward of the first indoor heat exchanger 50a. The upper end of the second indoor heat exchanger 50b is connected to the upper end of the first  
10 indoor heat exchanger 50a, and the first indoor heat exchanger 50a and the second indoor heat exchanger 50b are combined so as to form an inverted V shape when viewed from the sides. The second indoor heat exchanger 50b is disposed so as to cover the cross flow fan 71 from the upper central portion to the upper front side.

The third indoor heat exchanger 50c is disposed on the lower portion of the second indoor heat exchanger 50b so as to cover the front portion of the cross flow fan 71. The upper end of the third indoor heat exchanger 50c is connected at an angle to the lower end of the second indoor heat exchanger 50b, and an obtuse angle is formed by means of the third indoor heat exchanger  
15 50c and the second indoor heat exchanger 50b. The third indoor heat exchanger 50c is parallel with the height direction, i.e., the perpendicular direction, and is perpendicular with respect to the lower unit 7 that covers the horizontal surface of the lower portion of the indoor heat exchanger 50. In addition, the lower end of the third indoor heat exchanger 50c is the lower end of  
20 the indoor heat exchanger 50, and the lower end of the third indoor heat exchanger 50c, i.e., the lower end of the front side of the indoor heat exchanger 50, is positioned at approximately the same height as the central axis of the cross flow fan 71.

The fourth indoor heat exchanger 50d is disposed on the lower portion of

the first indoor heat exchanger 50a so as to cover the rear portion of the cross flow fan 71. The upper end of the fourth indoor heat exchanger 50d is connected at an angle to the lower end of the first indoor heat exchanger 50a, and an obtuse angle is formed by means of the fourth indoor heat exchanger 50d and the first indoor heat exchanger 50a. The fourth indoor heat exchanger 50d is parallel with the height direction, and is perpendicular with respect to the lower unit 7 that covers the horizontal surface of the lower portion of the indoor heat exchanger 50. In addition, the lower end of the fourth indoor heat exchanger 50d is the lower end on the rear side of the indoor heat exchanger 50, and the lower end of the fourth indoor heat exchanger 50d, i.e., the lower end of the rear side of the indoor heat exchanger 50, is positioned at approximately the same height as the central axis of the cross flow fan 71.

The third indoor heat exchanger 50c and the fourth indoor heat exchanger 50d have the same height in the height direction, and the upper and lower ends of the third indoor heat exchanger 50c and the fourth indoor heat exchanger 50d are positioned at the same height. Thus, the lower end of the front side and the lower end of the rear side of the indoor heat exchanger 50 have the same height, and are positioned at approximately the same height as the central axis of the cross flow fan 71. In addition, the lower ends on the front and rear sides of the indoor heat exchanger 50 extend from the lower ends on the front and rear of the inverted V shaped portion downward in the perpendicular direction to approximately the same height as the central axis of the cross flow fan 71.

The first indoor heat exchanger 50a, the second indoor heat exchanger 50b, the third indoor heat exchanger 50c, and the fourth indoor heat exchanger 50d are unitarily connected to form the indoor heat exchanger 50 by mutually fixing attachment plates that are arranged on both side ends (the ends in the vertical direction when viewed from the front). The indoor heat exchanger 50 has a cross-sectional shape that is a combination of the inverted V shaped portion that is formed by the first indoor heat exchanger 50a and the second

indoor heat exchanger 50b, and the straight portions that extend downward in the perpendicular direction from the respective lower ends of the first indoor heat exchanger 50a and the second indoor heat exchanger 50b. The indoor heat exchanger 50 has a linearly symmetrical cross-sectional shape from front to back with regard to a parallel line in the perpendicular direction that passes through the apex of the inverted V shape, and the first indoor heat exchanger 50a and the second indoor heat exchanger 50b, and the third indoor heat exchanger 50c and the fourth indoor heat exchanger 50d, are symmetrical from front to back. The indoor heat exchanger 50 is formed into a cross-sectional shape that includes a front to back symmetrical inverted V shape when viewed from the sides, as noted above, however, when viewed from the front, is rectangular in shape in the horizontal direction.

The auxiliary lines 51 connect the indoor heat exchanger 50 with the refrigerant line 4 that are on the exterior of the indoor unit 2, and allows refrigerant to flow back and forth between the indoor heat exchanger 50 and the outdoor heat exchanger 30. As shown in Figs. 4 and 6, the auxiliary lines 51 are connected to the heat transfer line of the indoor heat exchanger 50. Note that Fig. 6 is a plan view of the right side portion of the indoor unit 2 with the upper casing 6 removed. The auxiliary lines 51 project out from the right side of the indoor heat exchanger 50, and pass through the space on the right side of the indoor heat exchanger 50. The auxiliary lines 51 curve toward the back surface of the indoor unit 2 after projecting out from the right side of the indoor heat exchanger 50, and the plurality of the auxiliary lines 51 are gathered together and covered by a protective tube 53. The gathered auxiliary lines 51 extend downward in the space on the right side of the indoor heat exchanger 50 along the back surface of the indoor unit 2, further curve toward the left side surface of the indoor unit 2 in the lower rear space of the indoor unit 2, and are connected to the refrigerant line 4.

An auxiliary support member 52 is provided near both side surfaces of the

indoor heat exchanger 50, and as shown in Fig. 4, supports the indoor heat exchanger 50 from the inner side thereof. Because the indoor heat exchange unit 5 has an inverted V shape and opens downward, the indoor heat exchange unit 5 is placed from above on the lower unit 7 on which the cross flow fan 71 and the indoor fan motor 72 have already been installed, and is supported by the lower unit 7 via the auxiliary support member 52.

<Upper casing>

As shown in Figs. 3 and 5, the upper casing 6 forms the upper portion of the indoor unit 2, and is formed by an upper front surface 60, a top surface 61, and upper side surfaces 62, 63.

The upper front surface 60 covers the upper front side of the indoor unit 2, and covers the front of the indoor heat exchanger 50. The upper front surface 60 is formed to be substantially flat, and a stepped portion is provided on a portion thereof. A front surface intake port 601 composed of a long slit shaped opening in the longitudinal direction of the indoor unit 2 is provided in the upper surface of the stepped portion. The front surface intake port 601 is arranged toward the top of the indoor unit 2.

The top surface 61 covers the top surface of the indoor unit 2, and covers the upper portion of the indoor heat exchanger 50. Top surface intake portions 611 composed of a plurality of slit shaped openings are provided in the top surface 61. The top surface intake ports 611 are arranged from the front side to the rear side of the top surface 61, and have a larger intake area than the front surface intake port 601. Because of this, sufficient air is also drawn in from the rear side of the top surface of the indoor unit 2.

The upper side surfaces 62, 63 cover the upper portions of the side surfaces of the indoor unit 2, and cover the side portions of the indoor heat exchanger 50. The upper side surface 62, 63 is the upper right side surface 62 and the upper side surface 63 is the upper left side surface 63, and when viewed from the front, the upper right side surface 62 is disposed on the right side of the

indoor heat exchanger 50 and the upper left side surface 63 is disposed on the left side of the indoor heat exchanger 50.

In addition, the lower end of the upper casing 6 is formed horizontally, and by placing the upper casing 6 on the lower unit 7, the boundary between the upper casing 6 and the lower unit 7 will be a horizontal line, and will appear when viewed from the front and the sides of the indoor unit 2.

<Lower unit>

The lower unit 7 forms the lower portion of the indoor unit 2, and as shown in Figs. 7 and 8, the lower casing 70, the cross flow fan 71, the indoor fan motor 72, the electric component box 73, and the like are in modularized form.

[Lower casing]

The lower casing 70 is formed by a lower front surface 74, a bottom surface 75, lower side surfaces 76, 77, a support portion 78, and the like, and has a color that is different from that of the upper casing 6.

The lower front surface 74 is a component that visually appears as the lower front surface of the indoor unit 2 when viewed from the front, and the upper end thereof is disposed so as to incline on the front of the indoor unit 2. As shown in Fig. 3(a), the upper end of the lower front surface 74 is formed horizontally, and forms a horizontal border line together with the lower end of the upper casing 6. In addition, a discharge port 741 composed of an opening along the longitudinal direction of the indoor unit 2 is arranged in the lower front surface 74. As shown in Fig. 5, the discharge port 741 communicates with a space in the interior of the support portion 78 in which the cross flow fan 71 is accommodated, and the air flow produced by the cross flow fan 71 is discharged indoors through the discharge port 741. In addition, a horizontal flap 742 that guides the air discharged indoors is arranged in the discharge port 741. The horizontal flap 742 is rotatably arranged in the center of a shaft that is parallel in the longitudinal direction of the indoor unit 2, and can open and close the discharge port 741 by being rotatably driven by a flap motor (not shown in the

figures).

The bottom surface 75 covers the bottom surface of the indoor unit 2, and is formed to be flat. The bottom surface 75 is disposed horizontally, and the support portion 78 is disposed on top thereof.

5        The lower side surfaces 76, 77 are components that visually appear as lower side surfaces of the indoor unit 2 when viewed from the sides, and cover the lower side surfaces of the indoor unit 2. The lower side surface 76 is the lower right side surface 76 and the lower side surface 77 is the lower left side surface 77, and when viewed from the front, the lower right side surface 76 is  
10       disposed on the right side of the indoor unit 2 and the lower left side surface 77 is disposed on the left side of the indoor heat exchanger 50. In addition, the upper ends of the lower side surfaces 76, 77 are formed horizontally in the same way as the lower front surface 74. With the upper casing 6 placed on the lower unit 7, the lower end of the upper casing 6, and the upper ends of the lower front  
15       surface 74 and the lower side surfaces 76, 77 of the lower unit 7 meet to form a horizontal boundary line.

The support portion 78 is surrounded by the lower side surface 74, the bottom surface 75, and the lower side surfaces 76, 77, and the upper surface of the support portion 78 is positioned above the upper ends of the lower front  
20       surface 74 and the lower side surfaces 76, 77. The cross flow fan 71, the indoor fan motor 72, the electric component box 73, the indoor heat exchange unit 5, and the like are installed from above on the support portion 78, and the cross flow fan 71, the indoor fan motor 72, the electric component box 73, the indoor heat exchange unit 5, and the like are supported from below.

25       The support portion 78 supports the indoor heat exchanger 50 via the auxiliary support portion 52 of the indoor heat exchange unit 5. The upper surface of the support portion 78 is at approximately the same height as the central axis of the cross flow fan 71. As shown in Fig. 7, drain pans 781, 782 and a fan accommodation portion 787 are provided on the upper surface of the



support portion 78.

The drain pans 781, 782 are components that receive water drops that are produced on the surface of the indoor heat exchanger 50 during heat exchange, and are formed from concave members that are sunken downward from the upper surface of the support portion 78. A front drain pan 781 and a rear drain pan 782 are the drain pans 781, 782, and as shown in Fig. 5, the front drain pan 781 is disposed below the third indoor heat exchanger 50c, i.e., below the front lower end of the indoor heat exchanger 50. The rear drain pan 782 is disposed below the fourth indoor heat exchanger 50d, i.e., below the rear lower end of the indoor heat exchanger 50. The front drain pan 781 and the rear drain pan 782 are disposed to interpose the cross flow fan 71 from front to rear. The front drain pan 781 and the rear drain pan 782 are positioned at approximately the same height, and the bottom surface of the front drain pan 781 and the rear drain pan 782 are positioned lower than the height of the central shaft of the cross flow fan 71, but are disposed near the lower ends of the indoor heat exchanger 50. Note that the bottom surfaces of the front drain pan 781 and the rear drain pan 782 that receive the drain water are slightly inclined toward the right side of the indoor unit 2. Then, as shown in Fig. 8, a communicating portion 783 that connects the front drain pan 781 and the rear drain pan 782 is provided on the right side portion of the support portion 78, and a water removal hole 784 which penetrates the communication portion 783 downward is arranged. As shown in Fig. 9, the water removal hole 784 communicates with the interior of a drain hose 785 that serves to discharge drain water from the drain pans 781, 782 to the exterior. The drain water that drains from the indoor heat exchanger 50 is received by the front drain pan 781 and the rear drain pan 782, is collected by the communicating portion 783, and is discharged from the water removal hole 784 to the exterior of the device via the drain hose 785.

The fan accommodation portion 787 is a portion in which the cross flow fan 71 and the fan motor 72 are accommodated, and is arranged near the center

of the upper surface of the support portion 78. The fan accommodation portion 787 is formed by means of a member that is sunk downward into a semi-circular shape from the upper surface of the support portion 78, and accommodates the lower half of the cross flow fan 71 and the indoor fan motor 72. In addition, an  
5 air path that communicates with the accommodated cross flow fan 71 and the discharge port 741 is arranged in the interior of the support portion 78.

In addition, the support portion 78 includes a tongue portion 786 that projects upward from the upper surface of the support portion 78 between the rear drain pan 782 and the cross flow fan 71. The tongue portion 786 covers  
10 the rear portion of the cross flow fan 71, and the upper end of the tongue portion 786 is positioned at a somewhat lower height than the apex of the cross flow fan 71.

Although the front drain pan 781, the rear drain pan 782, and the fan accommodation portion 787 are arranged on the upper surface of the support  
15 portion 78, and the tongue portion 786 projects upward, other portions on the upper surface of the support portion 78 are formed to be flat and horizontal, and are positioned at approximately the same height as the central axis of the cross flow fan 71.

As noted above, the portion that is positioned at the highest position on  
20 the support portion 78 is the tongue portion 786, but the tongue portion 786 is positioned at the height of the apex of the cross flow fan 71 or lower. In addition, the upper surface of the support portion 78 is positioned above the upper surfaces of the lower front surface 74 and the lower side surfaces 76, 77. Because of this, each portion of the lower casing 70, including the support  
25 portion 78, is at the height of the apex of the cross flow fan 71 or lower.

Note that the back surface side of the upper surface of the support portion 78 is also at the height of the cross flow fan 71 or lower, but the portion between the top surface 61 of the upper casing 6 and the back surface side of the upper surface of the support portion 78 is closed by means of an installation plate that

is installed on an indoor wall surface (refer to Fig. 5). The installation plate 8 has a length that is approximately the same as the indoor heat exchanger 50 in the longitudinal direction of the indoor unit 2, and covers the back surface side of the indoor heat exchanger 50. The installation plate 8 forms, by covering the back surface side of the indoor unit 2, together with the upper casing 6, an air path through which air that exchanges heat with the indoor heat exchanger 50 passes, especially an air path of the back surface side.

[Cross flow fan]

The cross flow fan 71 is formed into a long, narrow cylindrical shape, and is disposed so that the central axis thereof is parallel with the horizontal direction. Blades are arranged on the circumferential surface of the cross flow fan 71, and the cross flow fan 71 will produce an air flow by rotating around the central axis. This air flow is taken in from the front surface intake port 601 and the top surface intake ports 611, and is an air flow that passes through the indoor heat exchanger 50 and is discharged indoor from the discharge port 741. The cross flow fan 71 is positioned in the approximate center of the indoor unit 2 when viewed from the sides. The cross flow fan 71 is supported by the support portion 78, and when supported, the upper half of the cross flow fan projects upward from the upper surface of the support portion 78.

[Indoor fan motor]

The indoor fan motor 72 rotatably drives the cross flow fan 71 around the central axis. The indoor fan motor 72 has a thin cylindrical shape having approximately the same diameter as the cross flow fan 71. As shown in Fig. 8, the indoor fan motor 72 is disposed in approximately the same axis as the cross flow fan 71 on the right side of the cross flow fan 71, and the indoor fan motor 72 has approximately the same height as the apex of the cross flow fan 71 when the indoor fan motor 72 is installed on the support portion 78 (refer to Fig. 7).

[Electric component box]

As shown in Figs. 6 and 8, the electric component box 73 accommodates

a control board 731 that serves to control the operation of the indoor unit 2. The electric component box 73 has a rectangular box shape, is disposed between the right lower surface 76 of the lower casing 70 and the support portion 78, and is positioned on the right side of the indoor heat exchanger unit 5. The electric component box 73 is installed and supported on the right side surface of the support portion 78 to the right of the indoor fan motor 72, and can be installed on the support portion 78 before the indoor heat exchange unit 5 is installed on the lower unit 7. In addition, the electric component box 73 is disposed toward the front side, and the space to the rear of the electric component box 73 is the space noted above through which the auxiliary lines 51 of the indoor heat exchange unit 5 pass. The electric component box 73 is disposed such that the strong electric components 732, such as the condenser, the power transistor, and the like which take up a large amount of space amongst the control components installed on the control board 731, are lined up in the axial direction with the indoor fan motor 72, and is disposed such that the indoor fan motor 72 and the electric component box 73 are behind one another when viewed from the sides. In addition, the upper surface of the electric component box 73 is positioned at approximately the same height as the apex of the indoor fan motor 72, i.e., the apex of the cross flow fan 71, when the electric component box 73 is supported on the lower casing 70.

Thus, the indoor fan motor 72, the electric component box 73, and all of the components of the lower casing 70 are positioned at the height of the apex of the cross flow fan 71 or lower when supported on the lower casing 70, and the lower unit 7 has an overall comparatively small dimensional shape in the height direction.

[Special characteristics]

[1]

With this indoor heat exchanger 50 that forms a part of the indoor unit 2 of the air conditioner 1, the third indoor heat exchanger 50c and the fourth indoor

heat exchanger 50d have the same length. When the total length (viewed from the sides) of indoor heat exchangers 50 are the same, the indoor heat exchanger 50 in which the lengths of the third indoor heat exchanger 50c and the fourth indoor heat exchanger 50d are the same will have a smaller maximum amount of error in the positions of the end portions of the indoor heat exchanger 50 due to error in the attachment angle than the indoor heat exchanger 50 in which the lengths of the third indoor heat exchanger 50c and the fourth indoor heat exchanger 50d are different.

For example, an assumed indoor heat exchanger 500 is shown in Fig. 10(b). Note that Fig. 10(a) shows the indoor heat exchanger 50 that forms a part of the indoor unit 2 of the air conditioner 1. The assumed indoor heat exchanger 500, like the indoor heat exchanger 50, is formed by four components, an eleventh indoor heat exchanger 500a, a twelfth indoor heat exchanger 500b, a thirteenth indoor heat exchanger 500c, and a fourteenth indoor heat exchanger 500d. The structure of each of the indoor heat exchangers 500a, 500b, 500c, and 500d are approximately the same as that of the indoor heat exchanger 50. However, the length of the thirteenth indoor heat exchanger 500c that forms the front end of the assumed indoor heat exchanger 500 is different from the length of the fourteenth indoor heat exchanger 500d that forms the rear end of the assumed indoor heat exchanger 500, and the thirteenth indoor heat exchanger 500c is longer. However, the total length of the thirteenth indoor heat exchanger 500c and the fourteenth indoor heat exchanger 500d is the same as the total length of the third indoor heat exchanger 50c and the fourth indoor heat exchanger 50d. Thus, the indoor heat exchanger 50 and the assumed indoor heat exchanger 500 have the same length when viewed from the sides, and have approximately the same surface area.

With this type of indoor heat exchanger 50 and assumed indoor heat exchanger 500, the impact of attachment error in each indoor heat exchanger on the position error of the lower ends of the indoor heat exchanger is considered

below. A schematic diagram of the lower front end of the indoor heat exchanger 50 is shown in Fig. 11(a), and a schematic diagram of the lower front end of the assumed indoor heat exchanger 500 is shown in Fig. 11(b). Note that in both figures, the indoor heat exchangers 50b, 50c, 500b, and 500c have been  
5 simplified by displaying them with straight lines in order to improve understanding.

When the third indoor heat exchanger 50c is connected to the lower end of the second indoor heat exchanger 50b at a perfectly correct angle, the third indoor heat exchanger 50c shown by the dotted and dashed line in the figure will be parallel with the vertical direction. However, in reality a certain amount of  
10 attachment error  $\alpha$  will be produced. Thus, the third indoor heat exchanger 50c is inclined at an angle  $\alpha$  with respect to the vertical direction. Even with the assumed indoor heat exchanger 500 shown in Fig. 11(b), the thirteenth indoor heat exchanger 500c is connected to the lower end of the twelfth indoor heat exchanger 500b at the same attachment angle error  $\alpha$ . Thus, the thirteenth  
15 indoor heat exchanger 500c is also inclined at an angle  $\alpha$  with respect to the vertical direction. Thus, when the indoor heat exchangers 50c, 500c are connected at an installation angle error  $\alpha$ , the greater the distance away from the connection point is, the greater each portion of the indoor heat exchangers 50c, 500c are separated from the ideal location shown by the dotted and dashed line  
20 in the figure. Thus, even if the attachment angle error  $\alpha$  is the same, the position error  $\Delta D2$  of the lower end of the thirteenth indoor heat exchanger 500c having a long length will be greater than the position error  $\Delta D1$  of the lower end of the third indoor heat exchanger 50c. In other words, the position error  $\Delta D2$  of the lower front end of the assumed indoor heat exchanger 500 will be greater  
25 than the position error  $\Delta D1$  of the lower front end of the indoor heat exchanger 50.

As noted above, the longer the component that forms the lower end of a heat exchanger is, the greater the position error of the lower end of the indoor heat exchanger. Said another way, the shorter the component that forms the

lower end of a heat exchanger is, the smaller the position error of the lower end of the indoor heat exchanger. Thus, if the total lengths of the indoor heat exchangers are the same, the indoor heat exchanger in which the lengths of the component that forms the lower front end and the component that forms the rear  
5 of the heat exchanger are the same will have a smaller maximum amount of position error in the lower ends than the indoor heat exchanger having different lengths. Then, with the indoor heat exchanger 50 in which the present invention has been applied, the length of the third indoor heat exchanger 50c and the fourth indoor heat exchanger 50d are the same. Because of this, with the  
10 indoor heat exchanger 50, the attachment angle error of the third indoor heat exchanger 50c and the attachment angle error of the fourth indoor heat exchanger 50d will have an impact on the position error of the lower ends of the indoor heat exchanger 50, but that impact will be comparatively small. Thus, with the indoor heat exchanger 50, the error tolerance of the attachment angle of  
15 the third indoor heat exchanger 50c, and the error tolerance of the attachment angle of the fourth indoor heat exchanger 50d, is relaxed. In addition, by relaxing the error tolerance of the attachment angles, the ability to assemble the indoor heat exchanger 50 will improve.

[2]

20 The indoor heat exchanger 50 that forms a part of the indoor unit 2 of the air conditioner 1 is formed by a component that has an approximate inverted V shape when viewed from the sides, and components that respectively extend downward from the lower ends of the front and rear portions having the approximate inverted V shape. Because of this, the positioning of the indoor  
25 heat exchanger 50 so as to cover the front, upper, and rear portions of the cross flow fan 71 can be simplified. Thus, the position of the indoor heat exchanger 50 will be comparatively low, and the indoor unit 2 will be reduced in size in the height direction. In addition, heat exchange efficiency is improved because the indoor heat exchanger 50 is disposed so as to surround the periphery of the

cross flow fan 71.

[3]

As noted above, when the indoor heat exchanger 50 is disposed so as to surround the periphery of the cross flow fan 71, the precise distance between each portion of the indoor heat exchanger 50 and the cross flow fan 71 will be important. Because of this, the present invention, in which the error tolerance of the attachment angle is relaxed as noted above, will be even more useful.

[4]

With the indoor unit 2 of the air conditioner 1, the indoor heat exchanger 50 is formed into a symmetrical shape from front to rear. Because of this, the indoor heat exchanger 50 can be formed by combining heat exchangers having a common shape from front to rear. Thus, the cost of manufacturing the indoor heat exchanger 50 can be reduced. More specifically, the first indoor heat exchanger 50a and the second indoor heat exchanger 50b, or the third indoor heat exchanger 50c and the fourth indoor heat exchanger 50d, can be respectively manufactured by means of fins which share a common shape, and thus costs can be reduced by the use of common components. In addition, not only the fins, but also the U shaped heat transfer lines that are installed on the sides of the first indoor heat exchanger 50a, the second indoor heat exchanger 50b, the third indoor heat exchanger 50c, and the fourth indoor heat exchanger 50d, can be commonly shared. Furthermore, the fixing plates that are installed on both sides of the first indoor heat exchanger 50a, the second indoor heat exchanger 50b, the third indoor heat exchanger 50c, and the fourth indoor heat exchanger 50d, can be commonly shared.

[5]

With the indoor unit 2 of the air conditioner 1, the indoor heat exchanger 50 is disposed so as to cover the front, upper, and rear portions of the cross flow fan 71, and both lower ends of the indoor heat exchanger 50 are positioned at approximately the same height as the central axis of the cross flow fan 71.



Thus, the indoor heat exchanger 50 is disposed in a comparatively low position in the interior of the indoor unit 2. Because of this, the size of the indoor unit 2 in the height direction will be reduced, and the indoor unit 2 will be reduced in size.

5 [6]

When the lower ends of the indoor heat exchanger 50 are disposed in a comparatively low position as noted above, the positions of the drain pans 781, 782 disposed below the indoor heat exchanger 50 will also be low. The drain water received in the drain pans 781, 782 is discharged to the exterior of the indoor unit 2 via the water removal hole 784 and the drain hose 785. With this configuration, if the difference in height between the drain hose 785 and the drain pans 781, 782 is large, the drain water can be discharged with good efficiency.

On the other hand, the upward movement of the drain pans 781, 782 is limited because the drain pans 781, 782 are disposed below the lower ends of the indoor heat exchanger 50. Thus, if the drain pans 781, 782 are disposed at different heights, the position of one of the drain pans will be lowered.

However, with the indoor unit 2 of the air conditioner 1, the drain pans 781, 782 are disposed at the same height. Because of this, the difference in the heights of the drain hose 785 and the drain pans 781, 782 can be maintained as large as possible. Thus, the drain water can be efficiently discharged.

[7]

With the indoor unit 2 of the air conditioner, the indoor heat exchanger 50 is formed in a shape that is symmetrical from front to rear, and the lower ends thereof from front to rear are the same height. When the positions of the lower front and rear ends of the indoor heat exchanger 50 are made different when viewed from the sides, one lower end will be lower and the other lower end will be higher. In this configuration, if the drain pans 781, 782 are disposed near the lower front and rear ends of the indoor heat exchanger 50, there will be a difference in height between the front drain pan 781 and the rear drain pan 782.

In addition, even when the positions of the lower front and rear ends of the indoor heat exchanger 50 are different, when the heights of the front drain pan 781 and the rear drain pan 782 are the same, there will be a large distance between the drain pan that is disposed below the lower of the two lower ends of the indoor heat exchanger 50 and the lower end of the indoor heat exchanger 50. In order to more reliably receive the drain water that drips down from the lower ends of the indoor heat exchanger 50 into the drain pans 781, 782, the drain pans 781, 782 are preferably disposed near the lower ends of the indoor heat exchanger 50.

In the indoor unit 2 of the air conditioner, the lower front and rear ends of the indoor heat exchanger 50 are at the same height. Because of this, the front drain pan 781 and the rear drain pan 782 are respectively disposed near the lower front and rear ends of the indoor heat exchanger 50, and the front drain pan 781 and the rear drain pan 782 are disposed at the same height. Thus, with the indoor unit 2 of the air conditioner 1, the drain pans 781, 782 can receive drain water more reliably, and the drain water can be discharged more efficiently.

[8]

With the indoor unit 2 of the air conditioner 1, the indoor heat exchanger 50 is formed into a symmetrical shape from front to rear. Because of this, the cost of manufacturing the indoor heat exchanger 50 can be reduced by using the same components on the front and rear of the heat exchanger 50. More specifically, the first indoor heat exchanger 50a and the second indoor heat exchanger 50b, or the third indoor heat exchanger 50c and the fourth indoor heat exchanger 50d, can be produced by means of fins having the same shape. In addition, not only the fins, but also the U shaped heat transfer lines that are installed on the sides of the first indoor heat exchanger 50a, the second indoor heat exchanger 50b, the third indoor heat exchanger 50c, and the fourth indoor heat exchanger 50d, can be commonly shared. Furthermore, the fixing plates that are installed on both sides of the first indoor heat exchanger 50a, the second

indoor heat exchanger 50b, the third indoor heat exchanger 50c, and the fourth indoor heat exchanger 50d, can be commonly shared.

[Other embodiments]

[1]

5           In the aforementioned embodiment, the indoor heat exchanger 50 has an approximate inverted V shape in cross-section when viewed from the sides. However, as shown in Fig. 12, an indoor heat exchanger 54 that has a V shape in cross-section when viewed from the sides is also possible. The indoor heat exchanger 54 is formed by a fifth indoor heat exchanger 54a, a sixth indoor heat exchanger 54b, a seventh indoor heat exchanger 54c, and an eighth indoor heat exchanger 54d. The fifth indoor heat exchanger 54a and the sixth indoor heat exchanger 54b form a component that has a V shaped cross-section when viewed from the sides. The seventh indoor heat exchanger 54c and the eighth indoor heat exchanger 54d form straight components that extend upward from the front and rear ends of the component having the V shape. Then, the seventh indoor heat exchanger 54c and the eighth indoor heat exchanger 54d are at the same height.

Even with this type of indoor heat exchanger 54, like in the aforementioned special characteristic (1), the error tolerance of the attachment angle of the seventh indoor heat exchanger 54c and the eighth indoor heat exchanger 54d is relaxed, and the ability to assemble the indoor heat exchanger 54 will improve.

In addition, this type of V shaped indoor heat exchanger 54 may be disposed to rotatably move.

25       [2]

In the aforementioned embodiment, the indoor heat exchanger 50 is formed by a component that has an inverted V shape when viewed from the sides, and straight components that extend downward from the lower front and rear ends of the component having the inverted V shape. However, the indoor heat

exchanger 50 may have a shape other than an inverted V shape. For example, the indoor heat exchanger 50 may have an arc shape, or may have an inverted U shape.

(INDUSTRIAL APPLICABILITY)

- 5        If the heat exchanger and the indoor unit of the air conditioner of the present invention are used, a reduction in heat exchangeability can be suppressed, and the error tolerance of the attachment angle of the heat exchanging units can be relaxed.